

Micro-cogeneration: State-of-art and R&D activities

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Fonds européen de développeme régional (FEDER) Europäischer Fonds für regionale Entwicklung (EFRE)



Baden-Württemberg





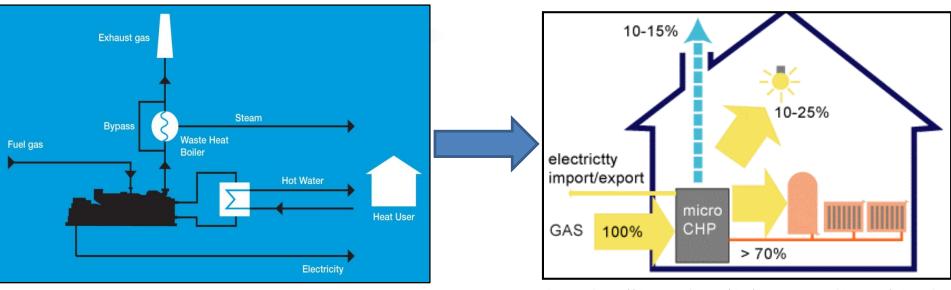


Agenda

- Introduction to micro cogeneration
 - System
 - Pros and cons
 - Prime mover technologies
 - Renewable energy based micro cogeneration
- Research result
- Conclusion and perspectives



What is "combined heat and power" (CHP) or "cogeneration"?

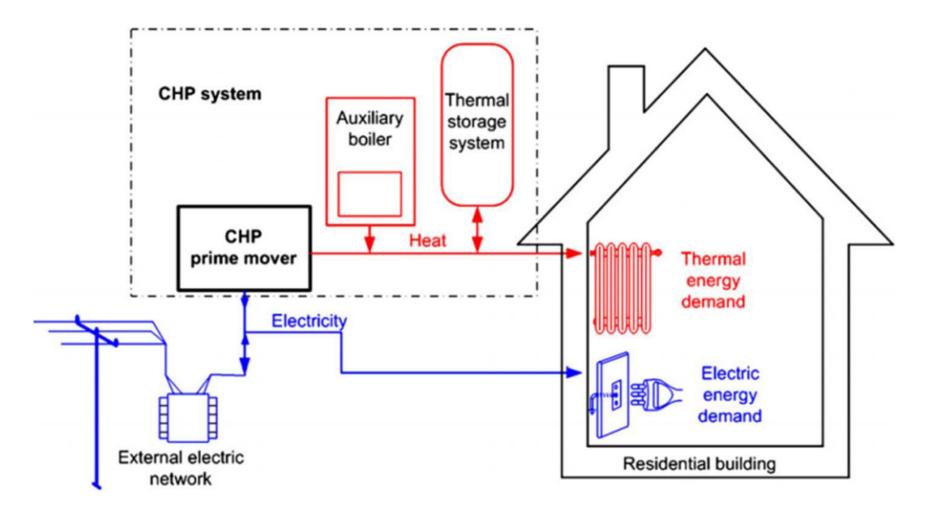


Source: https://www.clarke-energy.com/natural-gas/industrial/

Source: https://sites.google.com/site/fourseasonsarchitecture/micro-chp



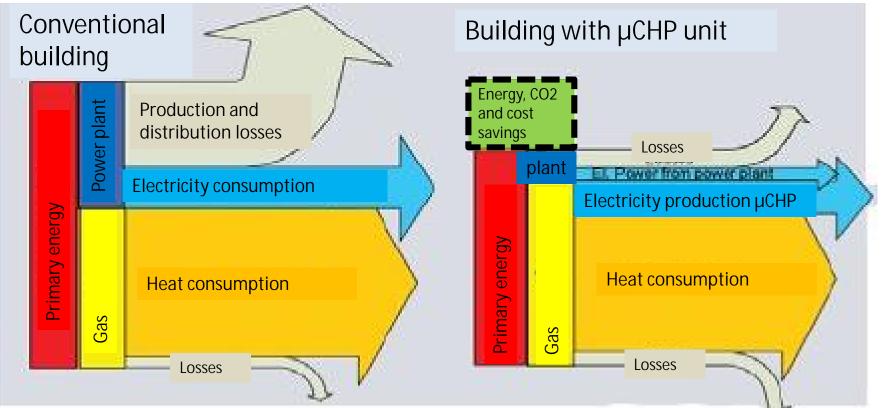
Typical integration of a micro CHP unit







Pro and cons of micro cogeneration



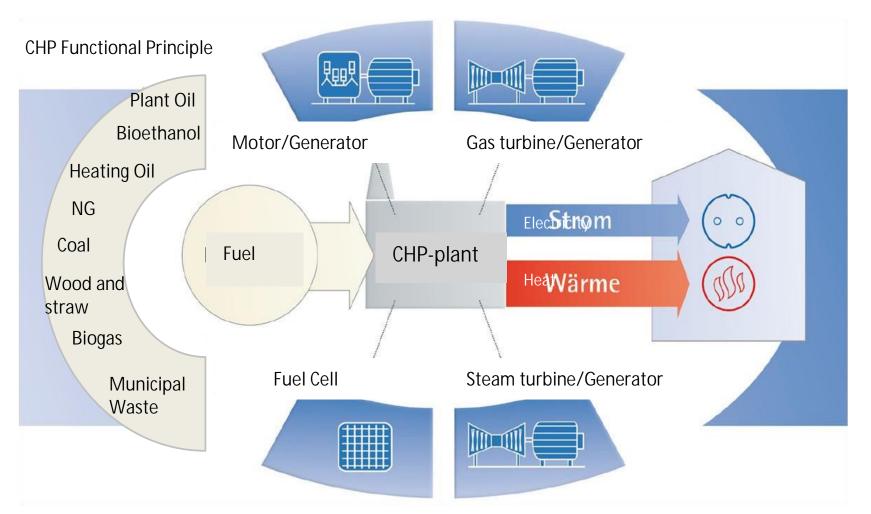
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- Flexibility in fuel use High investment costs Lack of the information +

- Life cycle of the techology

|3E

Prime mover technologies

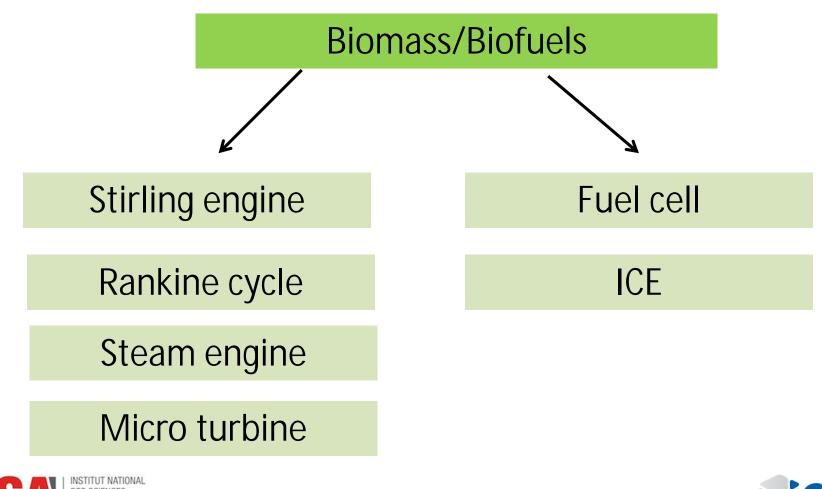


The German Association of Combined Heat and Power, http://www.bkwk.de/aktuelles/Broschur/Broschur_Internet.pdf

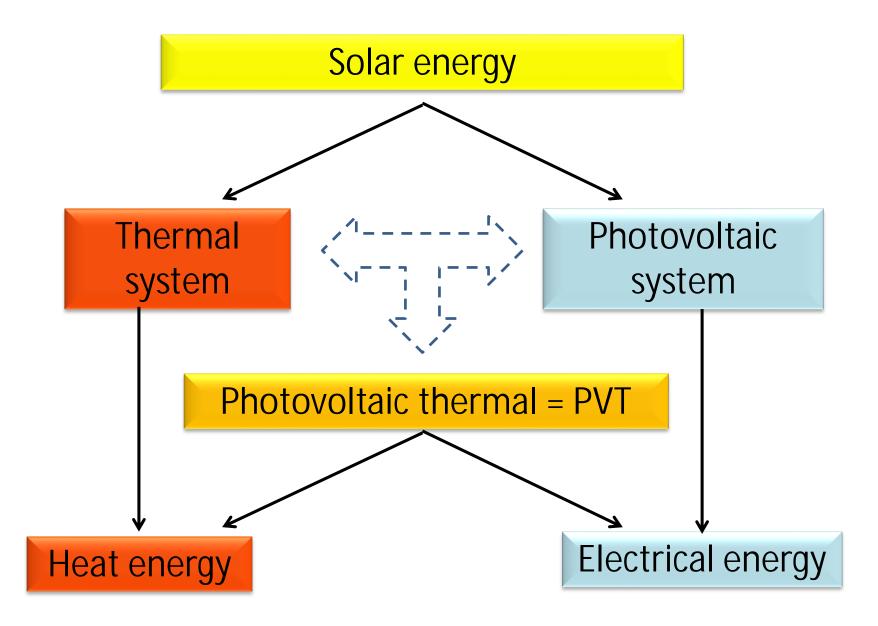




Micro-CHP systems using renewable energy



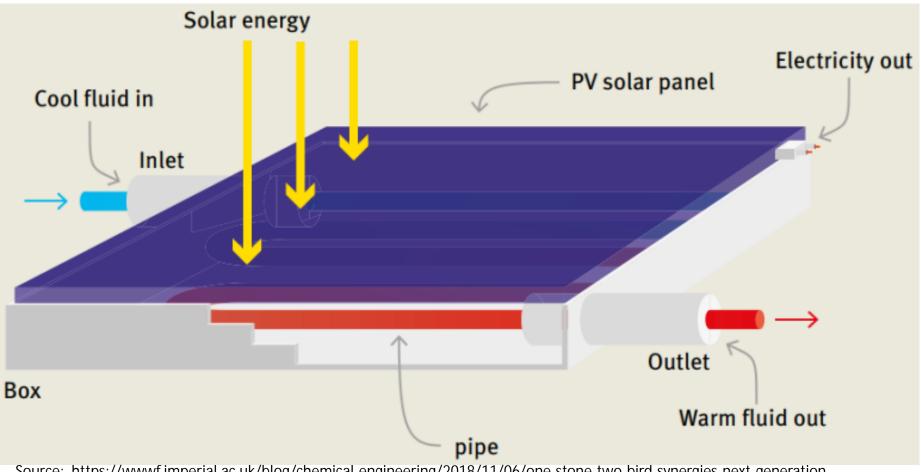








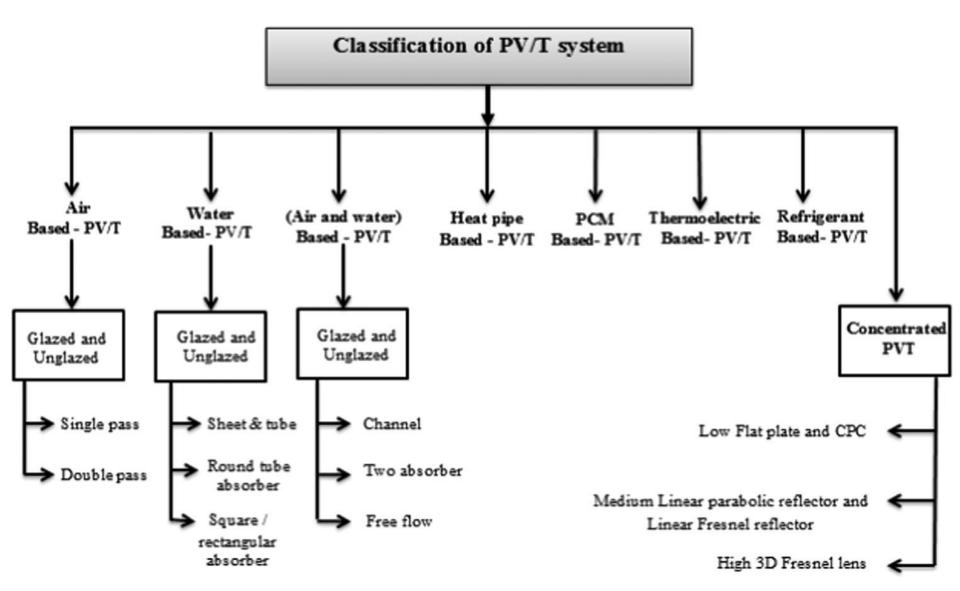
Water based flat plate PVT collector



Source: https://wwwf.imperial.ac.uk/blog/chemical-engineering/2018/11/06/one-stone-two-bird-synergies-next-generation-solar-technologies/







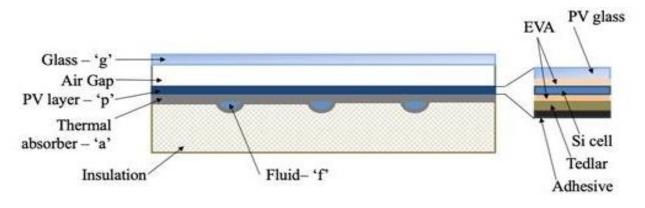
Source: A.M. Elbreki & all, The role of climatid-design-operational parameters on combined PV/T collector performance: A critical review





Modelling and simulation of a water based PVT collector

• Mathematical model of the PVT layers



- Energy and exergy analysis of the PVT operation under two different climate conditions
- Daily, weekly and yearly simulations of the PVT operation





Simplified energy balance of the PVT collector

PVT collector:

$$M_{pvt}C_{pv}\frac{dT_{pvt}}{dt} = Q_{losses} + Q_{pvt-f} + Q_{pvt} - E$$

Coolant fluid:

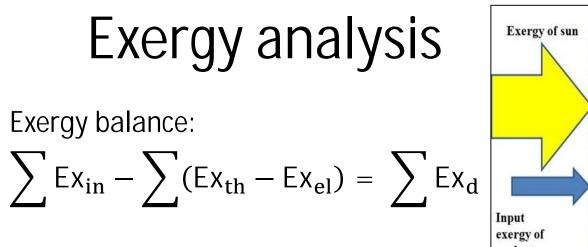
$$M_{f}C_{f}\frac{dT_{f}}{dt} = O_{f-pvt} + O_{f}$$

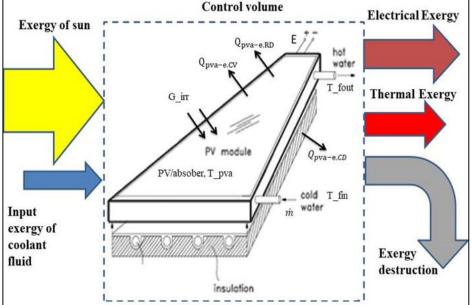
Assumptions:

- The temperature distribution is uniform
- no heat losses through the edges
- the optical and thermal properties of the materials and fluids are constant
- no surrounding shading or dust is taken into account









Exergy of solar irradiation:

$$\mathsf{Ex}_{in} = \mathsf{A}_{pva}\mathsf{N}_{c}\mathsf{G}_{irr}\left(1 - \frac{4}{3}\frac{\mathsf{T}_{0}}{\mathsf{T}_{sol}} + \frac{1}{3}\left(\frac{\mathsf{T}_{0}}{\mathsf{T}_{sol}}\right)^{4}\right)$$

Exergy of thermal energy:

$$Ex_{th} = \dot{m}c_{f}\left[(T_{out} - T_{in}) - T_{0}ln\frac{T_{out}}{T_{in}}\right]$$

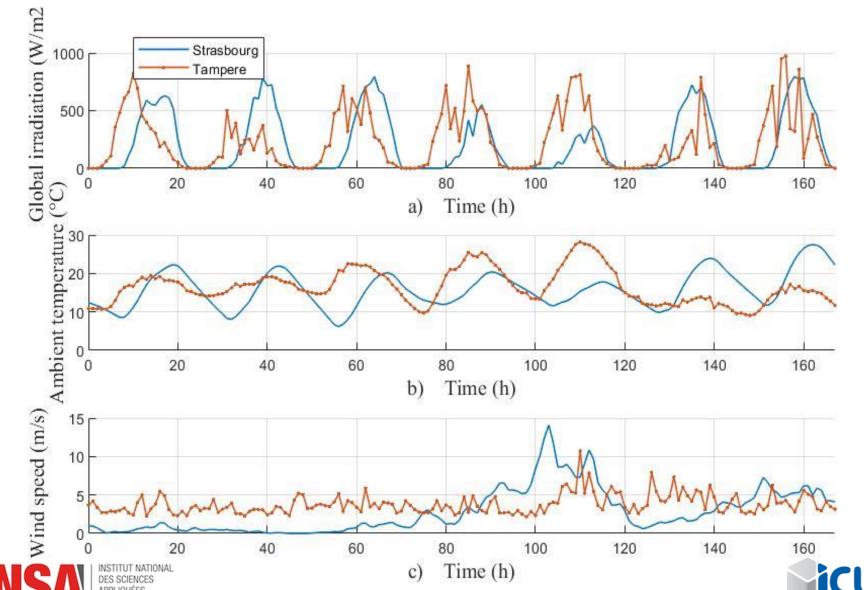
Exergy of electrical energy:

$$\mathsf{E} \mathsf{x}_{el} = \eta_{pv} \mathsf{G}_{irr} \mathsf{A}_{pva}$$

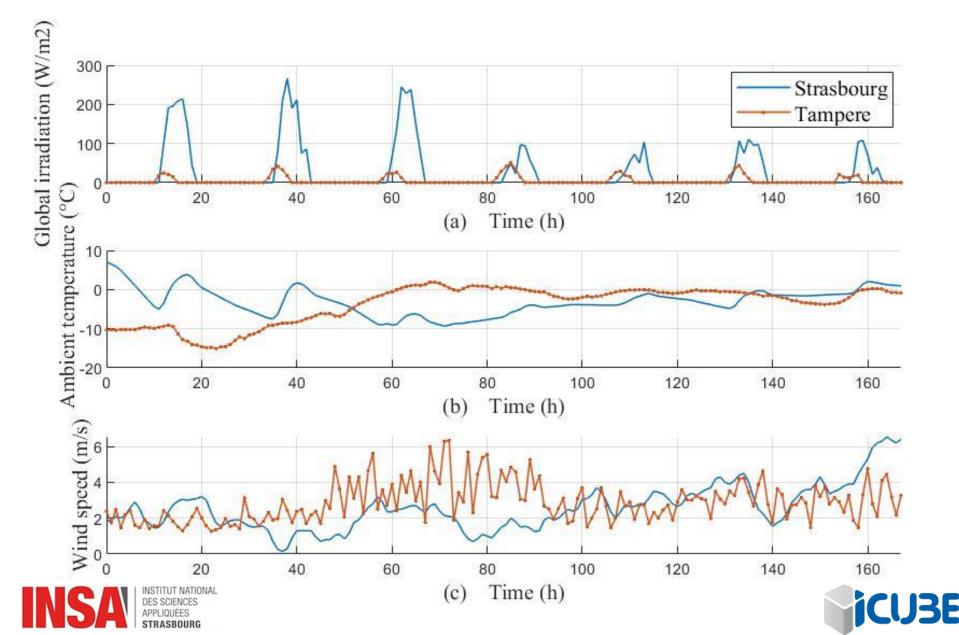




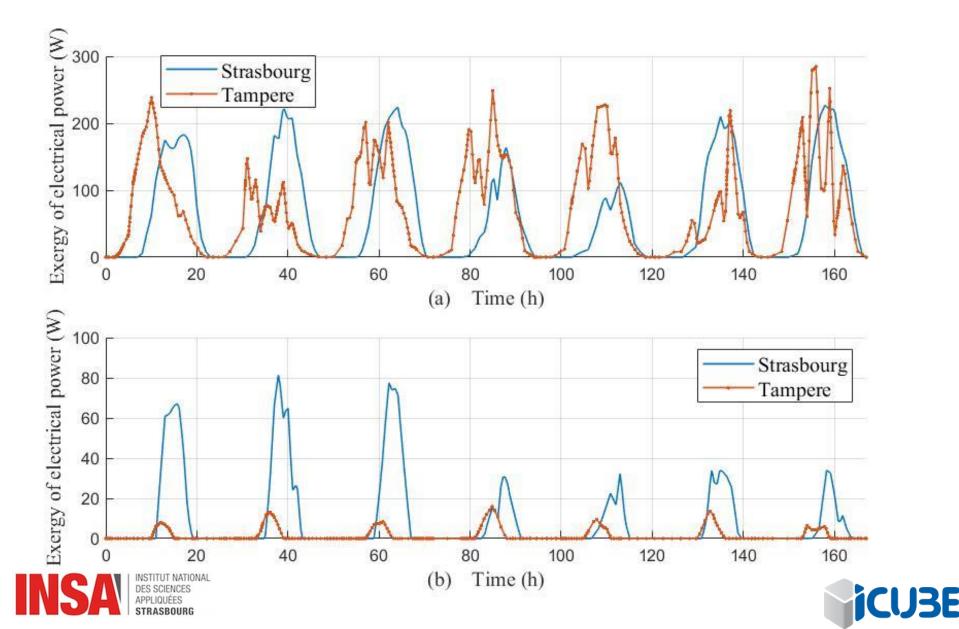
Two different climate conditions Tampere, Finland and Strasbourg, France



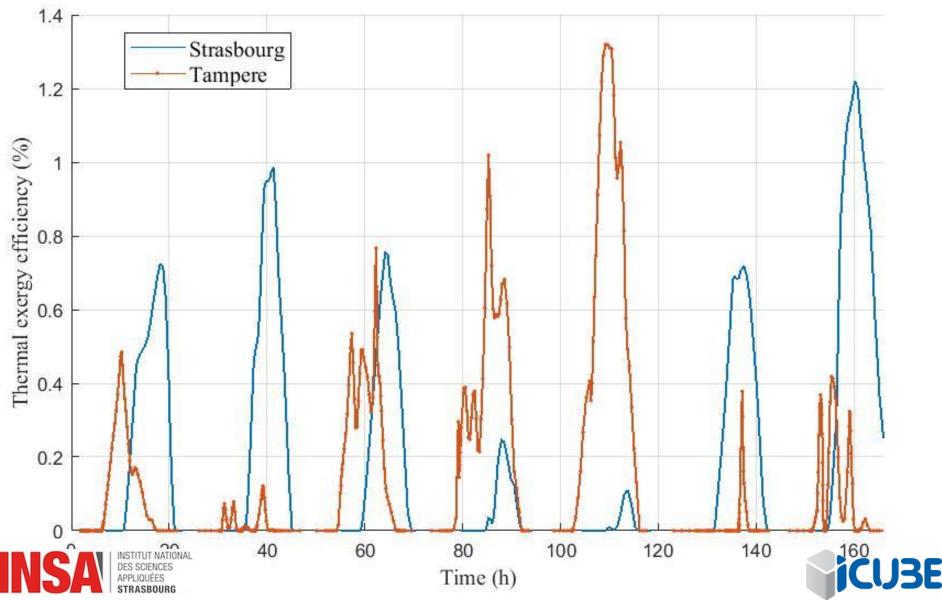
Winter week



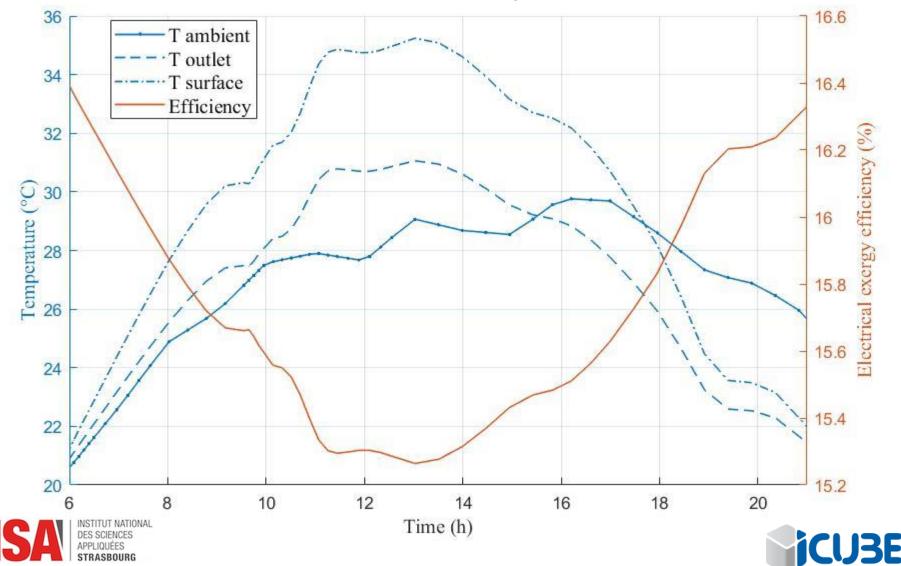
Simulation results

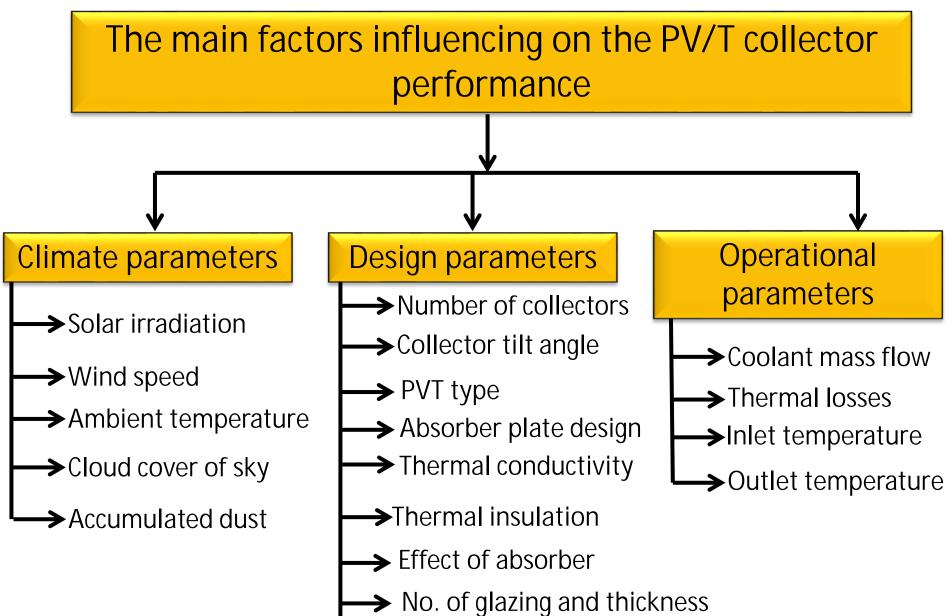


Simulation results Summer week



Simulation results Summer day

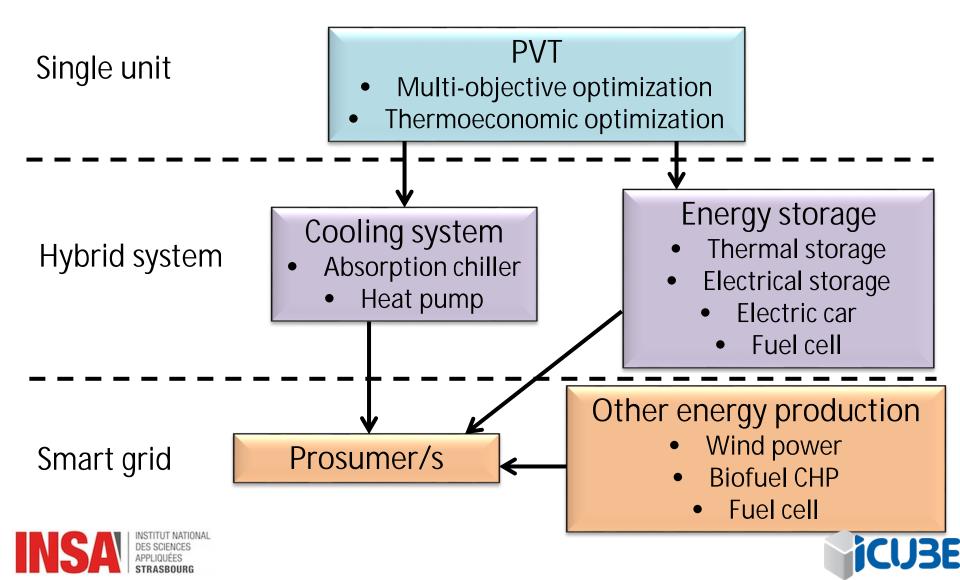




→ Area of collector



Conclusion and perspectives From a single unit to the smart grid



Conclusion and perspectives

- •Intelligent energy management system
 - → minimize the operation costs and power exchange between the main grid and smart grid
- •Co-operation with project partners
 - \rightarrow experimental study of the behaviour of a hybrid system including PVT







INSTITUT NATIONAL DES SCIENCES APPLIQUÉES **STRASBOURG**

Questions?

Thank you!







Dépasser les frontières projet après projet interreg wächst zusammen



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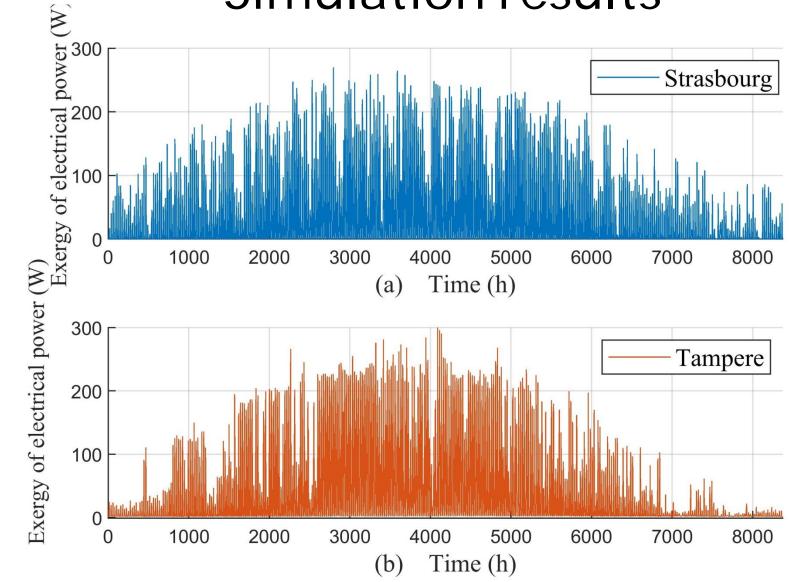








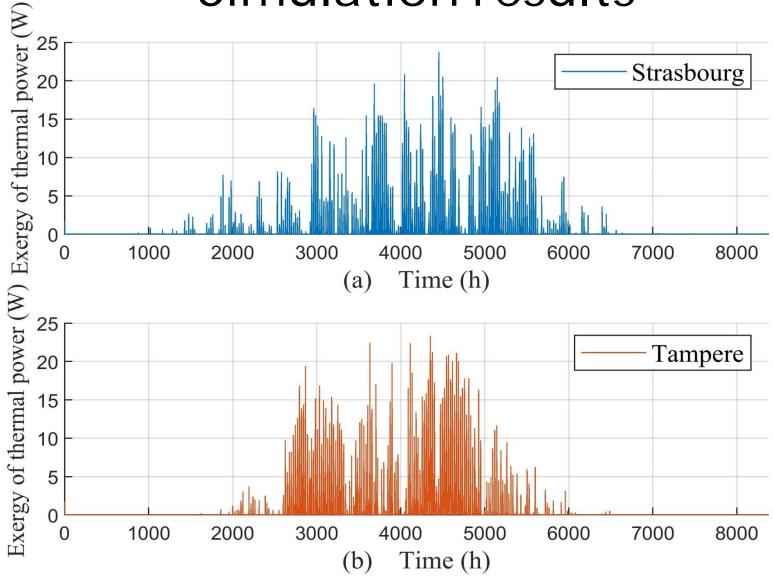
Simulation results







Simulation results







Parameters used in the model

The inlet temperature of the coolant fluid is 20 °C

*Table 1. The main geometrical, thermo-physical and optical properties of the PVT collector. With * the parameter varies with time.*

Property	Glass	Air gap	PV	Thermal absorber	Fluid	Insulation	Unit
Emissivity (ε)	0.9	-	0.96	-	-	-	-
Absorbance (α)	0.1	-	0.9	-	-	-	-
Transmittance (τ)	0.93	-	-	-	-	-	-
Thickness (H)	0.004	0.02	0.006	0.001	-	0.04	m
Area (A)	2	2	2	2	-	2	m^2
Mass flow	-	-	-	-	0.019	-	kg/s
Density (p)	2200	-	2330	2699	1050	16	kg/m3
Specific heat (c)	670	-	900	800	4180	1120	J/(kgK)
Thermal conductivity (k)	1.1	*	140	237	0.615	0.035	W/(mK)





Correlation for forced convection and relation for electrical efficiency

- Correlation for the heat transfer coefficient of forced convection
- depends on the wind speed

$$h_{pva-e,CV} = \begin{cases} 5.7 + 3.8v_{w}, & \text{for } v_{w} < \frac{5m}{s} \\ 6.47 + v_{w}^{0.78} & \text{for } v_{w} > \frac{5m}{s} \end{cases}$$

$$\eta_{\text{EL}(T)} = \eta_{\text{STC}}[1 - \beta_{\text{PV}}(T_{\text{pva}} - T_{\text{ref}})]$$





Sensitivity analysis

